

Fig. 1. Control system with one zero memory nonlinearity.

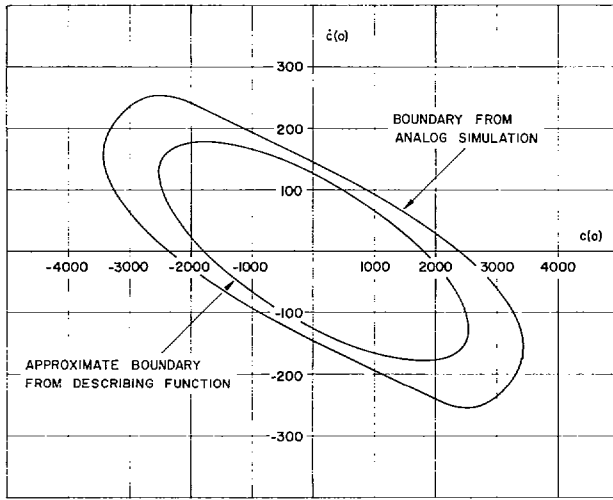


Fig. 2. Approximate and exact region of stable initial conditions for example problem.

With the assumption that  $N(e)$  is replaced by a linear element as in the preceding paragraph, the response  $e(t)$  can be calculated from knowledge of the initial conditions of the linear part;  $e(t)$  will contain an oscillatory component caused by the two poles on the  $j$  axis. The steady amplitude of oscillation can be found from the evaluation of the residues of the two neutrally stable poles.

The peak of the steady-state response of  $e$  is now set equal to the limit cycle amplitude read along the  $-N(e)$  locus at the  $-N(e), 1/G(j\omega)$  crossing. An equation results which contains all of the initial conditions of the linear-part differential equations, and which expresses a surface in the initial condition space. Every point on the surface of initial conditions produces steady-state oscillation in the system (subject to the approximation inherent in the describing function method and the degree with which the variable  $e$  is represented by the linear-

There is one crossing of  $1/G(j\omega)$  and  $-N(e)$  (which occurs on the negative real axis) with<sup>2</sup>

$$\begin{aligned} 1/G(j\omega) &= -N(e) = 0.1 \\ e &= 12.7 \\ \omega &= 0.1. \end{aligned} \tag{3}$$

Now set

$$a = 0.1e = -0.1c \tag{4}$$

and substitute in (1) taking the Laplace transform

$$e(s) = \frac{0.01s^2c(0) + (0.1s^2 + 0.01s)c(0)}{(s^2 + 0.01)(s + 0.1)} \tag{5}$$

where  $\ddot{c}(0)$  has been assumed zero in the interest of simplicity. By evaluating the residue at  $s = 0.1j$ , the peak steady state  $e(t)$  is found to be

$$\max \{e\} = \sqrt{\frac{1}{2} c^2(0)10^{-4} + \dot{c}^2(0)10^{-2} + c(0)\dot{c}(0)10^{-3}}. \tag{6}$$

ized system). A new surface of initial conditions is obtained for each crossing of the  $1/G(j\omega)$  and  $-N(e)$  loci.

The following example illustrates the method. Let the nonlinear element be saturation (with saturation occurring at one unit of  $e$ ), and let the linear part be expressed by

$$\begin{aligned} e &= \ddot{c} + 0.1\dot{c} + 0.01c \\ \ddot{c} &= -a. \end{aligned} \tag{1}$$

Then

$$G(s) = \frac{-e}{a}(s) = \frac{s^2 + 0.1s + 0.01}{s^3}. \tag{2}$$

By setting (6) equal to 12.7 [from (3)], the approximation to the boundary curve of stable initial conditions is obtained. A plot of the  $\dot{c}(0)$  vs.  $c(0)$  stability boundary approximation is shown in Fig. 2. The stability boundary as determined from analog simulation of (1) is shown for comparison.

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<sup>2</sup> *Ibid.*, p. 572.

### Forced Response of Linear Time-Varying Systems

The utility of error coefficients in studying the behavior of constant parameter linear systems has been well established [1]. This concept can be extended easily to time-varying linear systems in determining the response to any input, knowing  $H(s, t)$ , where  $H(s, t)$  is Zadeh's system function [2]-[4].

The general input-output relation of a time-varying linear system can be written as

$$L(p, t)e_2(t) = k(p, t)e_1(t) \tag{1}$$

where  $p = d/dt$ ,  $e_1(t)$  is the input and  $e_2(t)$ , the output. In other words,

$$L(p, t)W(t, \tau) = k(p, t)\delta(\tau) \tag{2}$$

where  $W(t, \tau)$  is the impulse response and  $\delta(\tau)$  the delta function. The method suggested here consists in considering the output of the system as a superposition integral, given by (for signals applied to  $t=0$ )

$$e_2(t) = \int_{-\infty}^t W(t, \tau)e_1(t - \tau)d\tau \tag{3}$$

and in expanding  $e_1(t - \tau)$  in terms of Taylor's series so as to get (assuming that the first  $n$ -derivatives of  $e_1(t)$  exist)

$$\begin{aligned} e_2(t) &= \int_{-\infty}^t W(t, \tau) \left[ e_1(t - \tau) - \tau e_1'(t) \right. \\ &\quad \left. + \frac{\tau^2}{2!} e_1''(t) \cdot \dots \right] d\tau. \end{aligned} \tag{4}$$

Since we are interested only in the forced response of the system, letting  $t \rightarrow \infty$ ,

$$\begin{aligned} e_2(t) &= \int_{-\infty}^{+\infty} W(t, \tau) \left[ e_1(t) - \tau e_1'(t) \right. \\ &\quad \left. + \frac{\tau^2}{2!} e_1''(t) \cdot \dots \right] d\tau. \end{aligned} \tag{5}$$

Let us now define

$$\begin{aligned} c_0 &= \int_{-\infty}^{+\infty} W(t, \tau)d\tau \\ c_n &= (-1)^n \int_{-\infty}^{+\infty} W(t, \tau)\tau^n d\tau. \end{aligned} \tag{6a}$$

Substituting from (6a), (5) may be written as

$$e_2(t) = c_0 e_1(t) + c_1 e_1'(t) + \dots \tag{6b}$$

which gives the forced response to any input  $e_1(t)$ . Also, by Zadeh's definition, the system function  $H(s, t)$  is given by [2]-[4]

$$H(s, t) = \int_{-\infty}^{+\infty} W(t, \tau)e^{-s\tau}d\tau \tag{7}$$

$$\begin{aligned} \lim_{s \rightarrow 0} H(s, t) &= \int_{-\infty}^{+\infty} W(t, \tau)d\tau \\ &= c_0 \end{aligned} \tag{8}$$

$$\begin{aligned} \lim_{s \rightarrow 0} \frac{\partial}{\partial s} H(s, t) &= - \int_{-\infty}^{+\infty} \tau W(t, \tau)d\tau \\ &= c_1 \end{aligned} \tag{9}$$

or in general

$$\lim_{s \rightarrow 0} \frac{\partial^n}{\partial s^n} H(s, t) = (-1)^n \int_{-\infty}^{+\infty} \tau^n W(t, \tau) d\tau = c_n. \tag{10}$$

Thus, given  $H(s, t)$  we can find the response to any forcing function  $e_1(t)$  using (6b) and (10). The coefficients  $c_0, c_1, \dots$  are all functions of time, and by comparison with their counterparts in the case of constant parameter linear systems, these may be called time-varying error coefficients. However, the use of these coefficients may not be very pronounced in the synthesis of linear time-varying systems.

Since  $c_0, c_1, \dots$  are all independent of  $e_1(t)$ , the forced response of any input may be quickly determined using (6b).

**Examples**

1) Let us consider a system, described by the differential equation,

$$\left[ p^2 + \frac{3 + 8e^t}{1 + 2e^t} p + \frac{12e^t + 12e^{2t} + 2}{(1 + 2e^t)^2} \right] e_2 = f(t).$$

First we shall find the coefficients  $c_0, c_1, \dots$  of this system and then find the forced response when  $f(t)$  is a unit step input, or a ramp input.

It can be readily checked that the  $H(s, t)$  of this system is

$$H(s, t) = \frac{1}{1 + 2e^t} \left[ \frac{1 + e^t}{s + 1} - \frac{1}{s + 2} - \frac{e^t}{s + 3} \right].$$

Hence

$$c_0 = \lim_{s \rightarrow 0} L_t H(s, t) = \frac{3 + 4e^t}{6(1 + 2e^t)}$$

$$c_1 = \lim_{s \rightarrow 0} \frac{\partial}{\partial s} L_t H(s, t) = -\frac{27 + 32e^t}{36(1 + 2e^t)}$$

Thus  $e_2(t)$  when  $f(t)$  is a unit step would be

$$e_2(t) = \frac{3 + 4e^t}{6(1 + 2e^t)}$$

and  $e_2(t)$  when  $f(t) = t$  would be

$$e_2(t) = \frac{3 + 4e^t}{6(1 + 2e^t)} - \frac{27 + 32e^t}{36(1 + 2e^t)}$$

The results may be checked by direct evaluation. The response to any input  $f(t)$  may be determined in a similar manner.

2) For the system given by

$$[t^2 p^2 + (3t^2 + 4t)p + (2t^2 + 6t + 2)] e_2(t) = f(t)$$

it can be easily checked that

$$H(s, t) = \frac{1}{t^2(s + 2)(s + 1)}$$

Hence

$$c_0 = \lim_{s \rightarrow 0} L_t H(s, t) = \frac{1}{2t^2}$$

$$c_1 = \lim_{s \rightarrow 0} \frac{\partial}{\partial s} L_t H(s, t) = \frac{-3}{4t^2}$$

$$c_2 = \lim_{s \rightarrow 0} \frac{\partial^2}{\partial s^2} L_t H(s, t) = \frac{7}{4t^2}$$

Thus when  $f(t) = t$

$$e_2(t) = \frac{1}{2t} - \frac{3}{4t^2} \quad \text{and for any other } f(t)$$

$$e_2(t) = c_0 f(t) + c_1 f'(t) + c_2 f''(t) + \dots$$

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**Tables of Laguerre Coefficients for Representation of Piecewise Linear Functions**

This correspondence describes a method of evaluating the Laguerre coefficients of a continuous (or piecewise continuous) function which can be approximated by a piecewise linear function. The function is viewed as a sum of ramp functions. Table I can be used to find the contribution of each ramp function to the Laguerre coefficients.

where  $U(t - t_k)$  is the unit step function and  $K_k$  is a coefficient of the ramp function ( $t - t_k$ ).

It is clear that the function  $c(t)$  is viewed as a sum of ramp functions. The approximation of a typical nonlinear characteristic is shown in Fig. 1. In Fig. 1,  $c(t)$  is approximated by

$$c(t) = tU(t) - (t - 0.2)U(t - 0.2). \tag{2}$$

One can convert such functions to standard polynomial form by having available a table of coefficients for approximating the function  $(t - t_k)U(t - t_k)$ . Such a table is used to find the contribution of each ramp function to the Laguerre coefficients.

If  $A_0, A_1, A_2, \dots, A_n$  represent the Laguerre spectra of  $c(t)$ , one may write

$$c(t) \cong \sum_{k=0}^n A_k L_k(t). \tag{3}$$

$L_k(t)$  is the  $k$ th normalized Laguerre polynomial and can be expressed as [3]

$$L_k(t) = \frac{1}{k!} e^t \frac{d^k}{dt^k} (t^k e^{-t}). \tag{4}$$

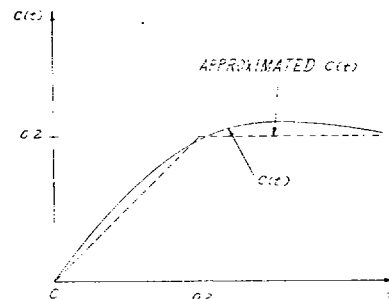


Fig. 1. The approximation of a nonlinear characteristic curve.

TABLE I

$A_k$	$t_i$	0.0	0.2	0.4	0.6	0.8	1.0
$A_0$		1.00000	0.81873	0.67032	0.54881	0.44933	0.36788
$A_1$		-1.00000	-0.98246	-0.93845	-0.87810	-0.80879	-0.73576
$A_2$		0.00000	0.01637	0.05363	0.09879	0.14379	0.18394
$A_3$		0.00000	-0.01528	-0.04648	-0.07910	0.01549	0.12263
$A_4$		0.00000	0.01439	0.04004	0.09393	0.14666	0.07664
$A_5$		0.00000	-0.17701	2.77646	0.30735	-0.09790	4.37469

**GENERAL**

In systems engineering it is sometimes advantageous to represent a functional relationship in terms of polynomials. However, one may encounter a functional relationship  $c(t)$  defined for  $0 < t < \infty$  which cannot be conveniently represented by a polynomial. In such cases, it is possible to approximate the functions by the piecewise linear representation [1]

$$c(t) \cong \sum_{k=0}^n K_k (t - t_k) U(t - t_k) \tag{1}$$

The characterizing coefficients  $A_k$  of the Laguerre spectrum of  $c(t)$  are [3]

$$A_k = \int_0^{\infty} c(t) L_k(t) e^{-t} dt \tag{5}$$

where  $e^{-t}$  is the weighting factor.  $c(t)$  is a continuous (or piece-wise continuous) function such as (3):

An alternative expression for (4) is [2]

$$L_k(t) = \sum_{j=0}^k \frac{(-1)^j}{j!} \binom{k}{j} t^j$$

$$= 1 - kt + \frac{k(k-1)}{4} t^2 - \dots$$

$$+ \frac{(-1)^k}{k!} t^k. \tag{6}$$

The first few Laguerre polynomials are given as follows

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